INF5490 RF MEMS

L11: RF MEMS resonators III

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Today's lecture

- Vertical vibrating resonators
  - Clamped-clamped beam (c-c beam)
    - Working principle
    - → **Detailed modeling**
  - free-free beam (f-f beam)
- Other resonator types
  - Tuning fork
  - Beam with lateral displacement
  - Disk resonators

Beam resonator

- How to obtain a higher resonance frequency than that which is possible with the comb-structure?
  - Mass should be reduced more -> **beam resonator**

- Beam resonator benefits
  - Smaller dimensions
  - Higher resonance frequency
  - Simple
  - Many frequency references on a single chip
  - Frequency variation versus temperature is more linear over a broader temperature range
  - Integration with electronics possible → lower cost

Beam resonator

First-order resonant frequency:

$$f_r = 1.03\sqrt{\frac{E}{\rho}}\,\frac{t}{L^2}$$

E = Young's modulus
ρ = Density
t = Beam thickness
L = Beam length

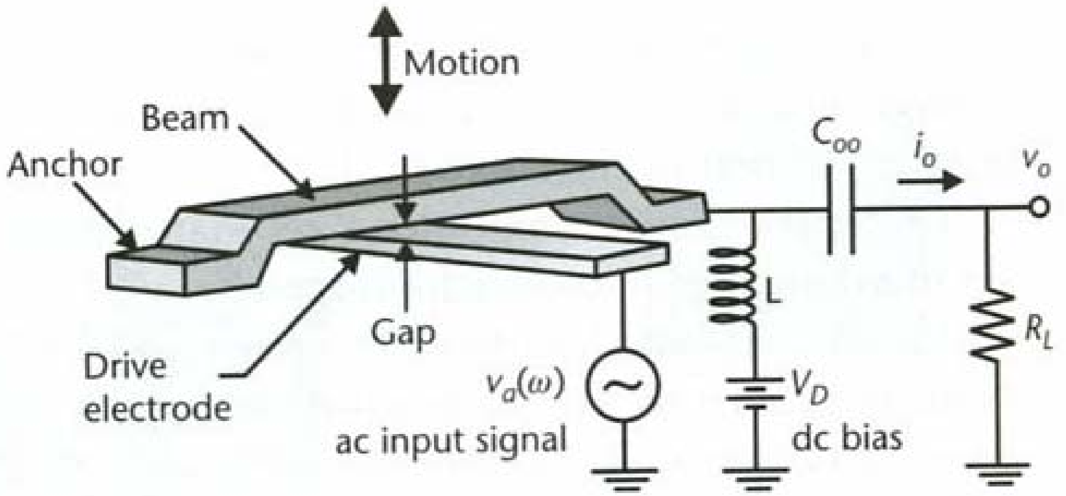


Figure 7.10 Illustration of a beam resonator and a typical circuit to measure the signal. The beam is clamped on both ends by anchors to the substrate. The capacitance between the resonant beam and the drive electrode varies with the deflection.

"One-port"-implementation

Output circuit

- Resonator is a time varying capacitance C(ω)
- Simple electrical output circuit
  – L = shunt RF blocking inductor: **Open** at high frequencies
  – C_∞ = series DC blocking capacitance: **Short circuited** at high frequencies
  – When Vd is a large DC-voltage bias, the dominating output current at frequency ω is given by: $i_o = V_d * dC/dt$
  – At high frequencies the current i_o is flowing through R_L
    - R_L may be the input impedance in the measurement equipment. Can be replaced by a transimpedance amplifier

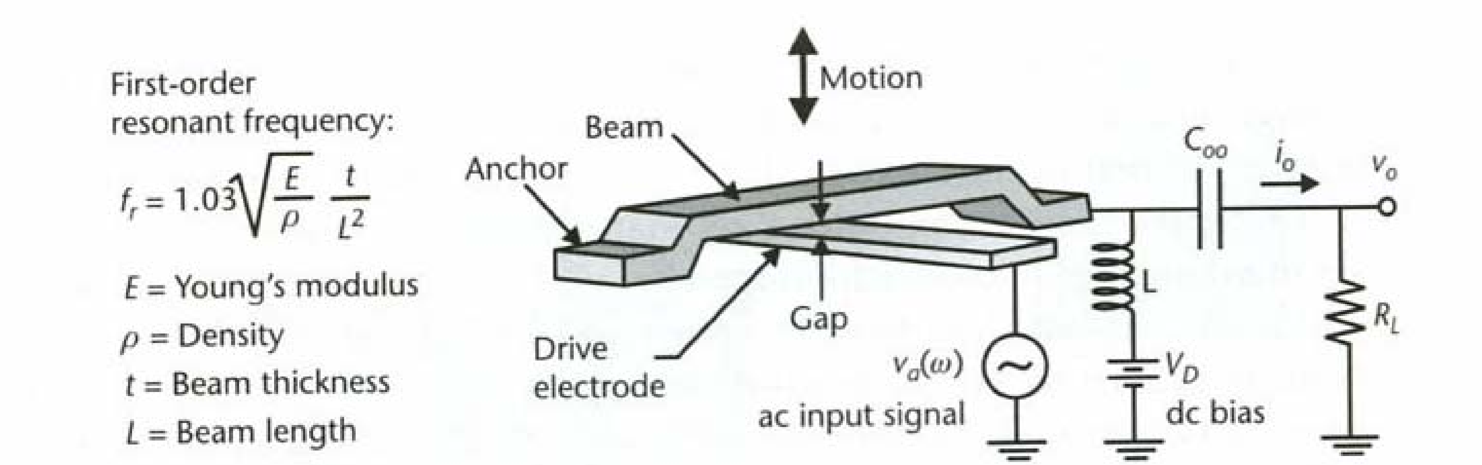


Figure 7.10 Illustration of a beam resonator and a typical circuit to measure the signal. The beam is clamped on both ends by anchors to the substrate. The capacitance between the resonant beam and the drive electrode varies with the deflection.

Mechanical resonance frequency

$$f_o = \frac{1}{2\pi}\sqrt{\frac{k_r}{m_r}} = 1.03\kappa\sqrt{\frac{E}{\rho}\frac{h}{L_r^2}}\left[1 - g(V_P)\right]^{1/2}, \qquad (12.2)$$

- Parameters
  - E = Youngs modulus
  - ρ = density of material
  - h = beam thickness
  - Lr = beam length
  - g models the effect of an **electrical spring constant k_e**
    - Is present when a voltage is applied between the electrodes
    - Subtracted from the mechanical spring constant, k_m (“beam-softening”)
  - κ =scaling factor (influenced by the surface topography, typical 0.9)
  - V_p = DC bias on conducting beam
  - k_r = effective resonator spring constant
  - m_r = effective mass
- **NB! E and ρ in expression + spring stiffness compensation term**

Typical frequencies

TABLE 12.1. μMechanical Resonator Frequency Design[a]

Frequency (MHz)	Material	Mode	h_r (μm)	W_r (μm)	L_r (μm)
70	Silicon	1	2	8	14.54
110	Silicon	1	2	8	11.26
250	Silicon	1	2	4	6.74
870	Silicon	2	2	4	4.38
870	Diamond	2	2	4	8.88
1800	Silicon	3	1	4	3.09
1800	Diamond	3	1	4	6.16

[a] Determined for free-free beams using Timoshenko methods that include the effects of finite h and W_r [11].

"Beam-softening"

- DC-voltage, Vd, will give a downward-directed electrostatic force
- This force opposes the mechanical restoring force of the beam
- The result is a lower **effective** mechanical spring constant

  – Resonance frequency decreases by a given factor $\sqrt{1 - C \cdot V_P{}^2 / (k \cdot g^2)}$

  – → **electrical tuning of resonance frequency!**

Detailed modeling

- **c-c beam** modeled as in the book
  - *T. Itoh et al: RF Technologies for Low Power Wireless Communications", chap. 12: "Transceiver Front-End Architectures Using Vibrating Micromechanical Signal Processors", by Clark T.-C. Nguyen*
  - *(+ summary from various publications)*

Clamped-clamped beam

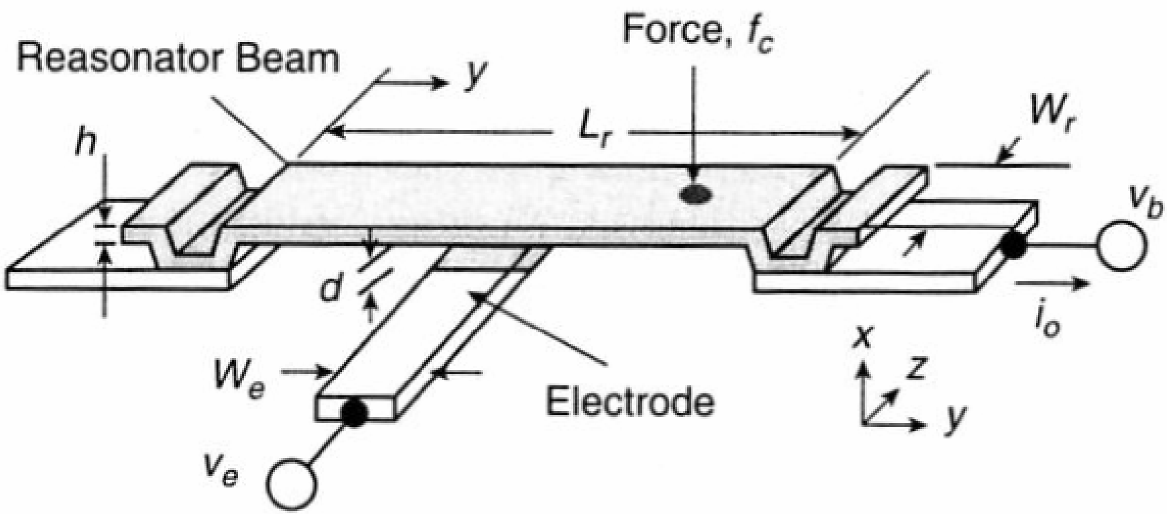


Figure 12.4. Perspective-view schematic of a clamped–clamped beam μmechanical resonator in a general bias and excitation configuration.

Calculating electrical excitation

- Two bias voltages are applied
- A) First calculate potential energy
- B) Calculate force →

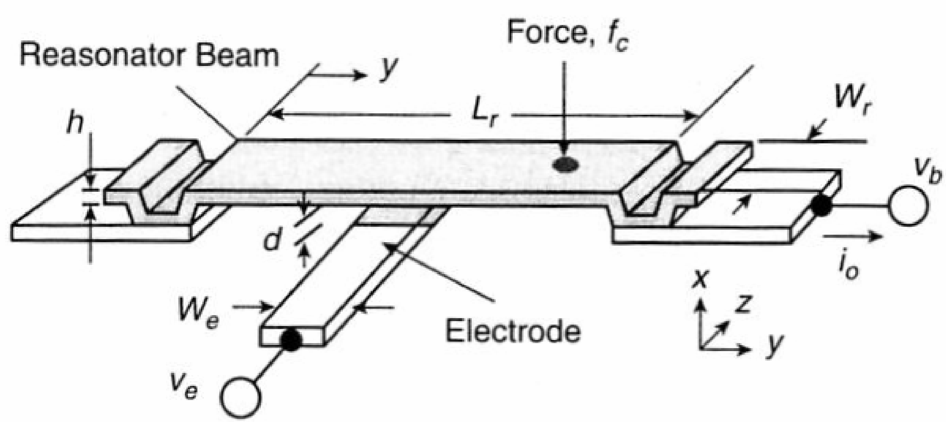


Figure 12.4. Perspective-view schematic of a clamped–clamped beam μmechanical resonator in a general bias and excitation configuration.

A. Electrical excitation

$$v_e = \text{input on electrode}$$

$$v_b = \text{input on beam}$$

$$v_e - v_b = \text{effective voltage}$$

$$U = \frac{1}{2}CV^2 = \frac{1}{2}C(v_e - v_b)^2 = \text{potential energy}$$

B. Force is change of potential energy vs. x

$$F_d = \frac{\partial U}{\partial x} = \frac{1}{2}(v_e - v_b)^2 \frac{\partial C}{\partial x}$$

$$= \frac{1}{2}(v_b{}^2 - 2v_b v_e + v_e{}^2)\frac{\partial C}{\partial x}$$

$$C = \frac{\varepsilon_0 A}{d_0} = \varepsilon_0 \frac{W_e W_r}{d_0}$$

$$W_e = \text{electrode width}, \mathrm{W_r} = \text{beam width}$$

$$d_0 = \text{electrode} - \text{resonator gap (static, non-resonance)}$$

$$\varepsilon_0 = \text{permittivity in vacuum}$$

Procedure, contd.

- C) Apply DC bias, V_p
- D) Calculate the force
- E) Discussion of different contributions
  - Off-resonance DC-force
  - Force with the same frequency as input voltage
  - Double frequency term

C. A DC voltage is applied to the beam

$$v_b = V_P, \ \ v_e = v_i = V_i \cos \omega_i t$$

D.

$$F_d = \frac{1}{2}(V_P{}^2 - 2V_P V_i \cos \omega_i t + V_i{}^2 \cos^2 \omega_i t) \frac{\partial C}{\partial x}$$

Observe that:

$$\cos^2 \omega_i t = \frac{1}{2}(1 + \cos 2\omega_i t)$$

$$V_i{}^2 \cos^2 \omega_i t = \frac{V_i{}^2}{2}(1 + \cos 2\omega_i t)$$

Then

$$F_d = (\frac{1}{2}V_P{}^2 - V_P V_i \cos\omega_i t + \frac{1}{2}\frac{V_i^2}{2} + \frac{1}{2}\frac{V_i^2}{2}\cos 2\omega_i t)\frac{\partial C}{\partial x}$$

E.

$$F_d = \frac{\partial C}{\partial x}(\frac{V_P{}^2}{2} + \frac{V_i^2}{4}) - V_P\frac{\partial C}{\partial x}V_i\cos\omega_i t + \frac{\partial C}{\partial x}\frac{V_i^2}{4}\cos 2\omega_i t$$

Off-resonance DC force
Static bending of beam

Force driven by the input frequency,
amplified by V_P

$$\frac{\partial C}{\partial x}\frac{V_i^2}{4}\cos 2\omega_i t$$

This term can drive the beam into vibrations at

$$2\omega_i = \omega_0, \ \ and \ \ \omega_i = \frac{\omega_0}{2}$$

The term can usually be neglected

Procedure, contd.

- → The main contribution to the force is proportional to cos
  - Drives beam into resonance
- F) Force gives displacement (x-variation)
  - The local spring constant varies over the width of the drive-electrode
  - Local displacement depends on the y position
- G) Derivation of an expression for the displacement, x(y), versus the spring constant at position y

Topology

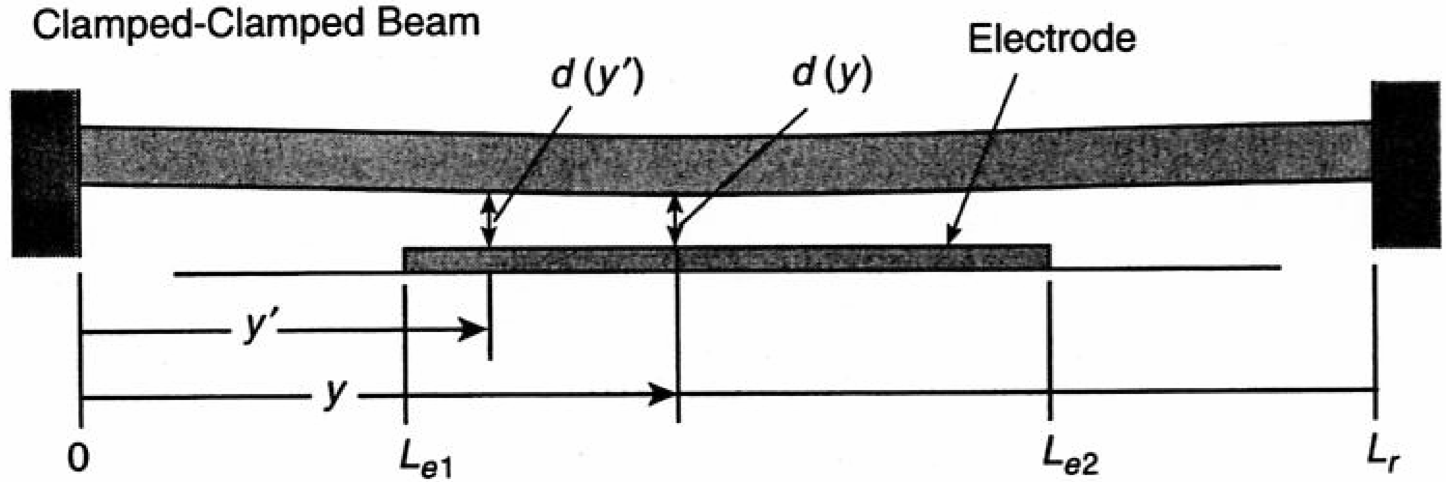


Figure 12.9. Resonator cross-sectional schematic for frequency-pulling and impedance analysis.

The main contribution to the force: $-V_P \frac{\partial C}{\partial x} V_i \cos \omega_i t$

At resonance the force will be:

$$F_d = -V_P \frac{\partial C}{\partial x} v_i(\omega_0)$$

The force will give a **varying displacement**, and the distance between the beam and electrode is dependent on y-position

$$\text{Generally}: F = k \cdot x, \text{static!}$$

$k(y) = k_{reff}(y)$ = effective beam stiffness in y

Dynamic performance of a mechanical system:

F.

$$H(s) = \frac{x}{F} = \frac{displacement}{force} = \frac{1/m}{s^2 + \frac{b}{m}s + \frac{k}{m}}$$

$$H'(s) = \frac{kx}{F} = \frac{k/m}{s^2 + \frac{b}{m}s + \frac{k}{m}} = \frac{\omega_0{}^2}{s^2 + \frac{\omega_0}{Q}s + \omega_0{}^2}$$

$$H'(j\omega_0) = \frac{\omega_0{}^2}{-\omega_0{}^2 + j\frac{\omega_0}{Q}\omega_0 + \omega_0{}^2} = \frac{Q}{j}$$

$kx = F \cdot \frac{Q}{j}$, at resonance (generally)

G. In our case:

$$x(y) = +\frac{Q}{j}\frac{F_d}{k_{reff}(y)} = -\frac{Q}{jk_{reff}} \cdot V_P \cdot \frac{\partial C}{\partial x} \cdot v_i$$

Force and displacement in opposite directions

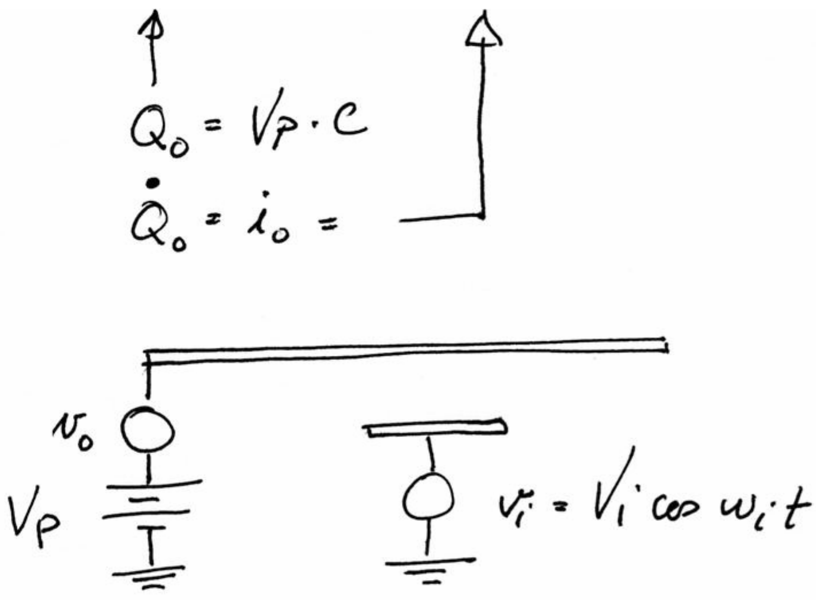
Procedure, contd.

- When the beam moves a time varying capacitance is established between the electrode and resonator

- H) This gives an output current that is ”DC-biased” via Vp
  - dC/dx is a non-linear term
  - dx/dt is speed

When the beam moves, a time dependent capacitance between the electrode and resonator will be created, giving an output current:

H. $$i_O = -V_P \frac{\partial C}{\partial x}\frac{\partial x}{\partial t} = \dot{Q}_O, \text{ where } Q_O = V_P C$$



Frequency response

- Typical parameters, Q, vacuum
  - Bandpass filter characteristics, Q ~ 10,000
  - Suitable for low loss reference oscillators and filters
- Q ~ a few hundreds at 1 AMP

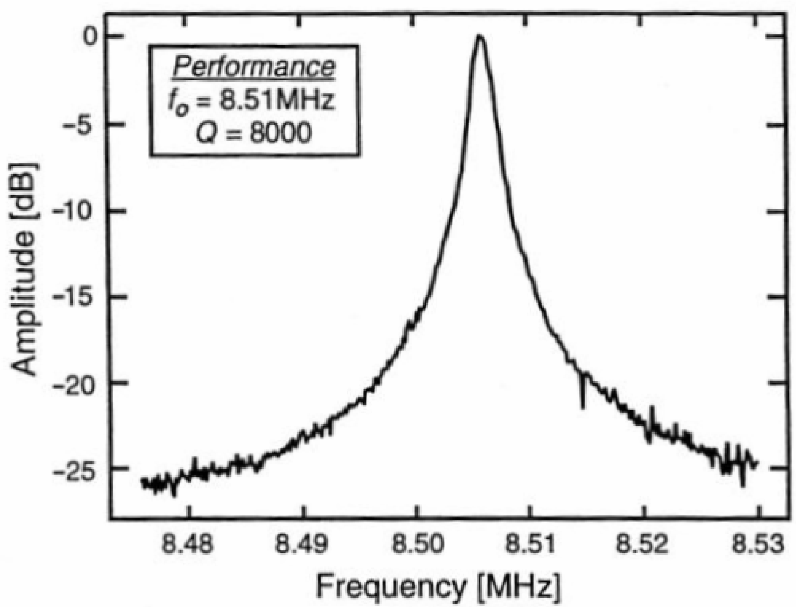


Figure 12.7. Frequency characteristic for an 8.5 MHz clamped–clamped beam polysilicon μmechanical resonator measured under 70 mtorr vacuum using a dc-bias voltage $V_P = 10$ V, a drive voltage of $v_i = 3$ mV, and a transresistance amplifier with a gain of 33 KΩ to yield an output voltage v_o. Amplitude $= v_o/v_i$. (From reference [18])

Procedure, contd.

- Transform to mechanical equivalent circuit:
  - ”mass-spring-damper”-circuit
  - NB! Still in the mechanical domain
- Beam described using ”lumped elements”
- Element values depend on position on beam, - dependent on y

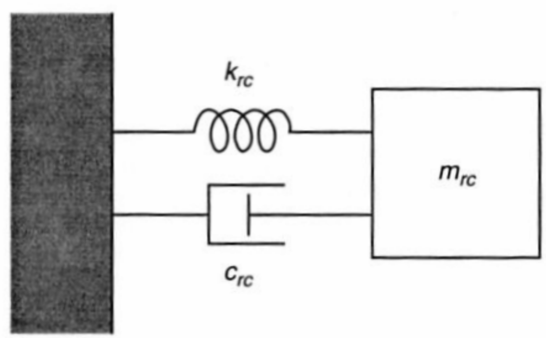


Figure 12.8. Lumped-parameter mechanical equivalent circuit for the micromechanical resonator of Figure 12.4.

I. Calculation of ”equivalent mass” as function of y

From R. A. Johnson: ”Mechanical Filters in Electronics”, Wiley, 1983

Simplified derivation of deflection equation

Form of ”fundamental mode”

Each point, y, has a specific effective mass, a specific velocity and spring constant

Lowest ”mass” in the middle, where the speed is maximum

$$m_r(y) = \frac{KE_{tot}}{\frac{1}{2}\left[v(y)\right]^2}$$

The equivalent mass at position y

KE_{tot} = peak kinetic energy of the system

$v(y)$ = velocity at location y

Flexural mode resonator: beam

$w =$ width, u = displacement in x – direction

$E =$ elastic modulus, $\rho =$ density

$I = \dfrac{wt^3}{12} =$ moment of inertia

The beam equation

$$\frac{\partial^2 u}{\partial t^2} = \frac{EI}{\rho A} \cdot \frac{\partial^4 u}{\partial y^4}, \text{ where } u = u_1 e^{j\omega t}$$

$$\Rightarrow \frac{\partial^4 u}{\partial y^4} = (\omega^2 \frac{\rho A}{EI})u$$

Trial solution :

here: k is the ”wave number”

$$u(y) = A\cosh ky + B\sinh ky + C\cos ky + D\sin ky$$

A, B, C, D can be found from initial conditions

Mode shape for fundamental frequency, c - c beam :

$$u(y) = \xi(\cos ky - \cosh ky) + (\sin ky - \sinh ky)$$

(From ”Johnson”)

Velocity in y - direction (along the beam)

$$v(y) = \dot{u}(y) = \frac{\partial}{\partial t}(u_1 e^{j\omega t}) = j\omega \cdot u(y)$$

Equivalent mass :

$$M_{eq}(y) = \frac{KE_{tot}}{\frac{1}{2}v^2(y)} = \frac{\frac{1}{2}\rho A \int_0^l v^2(y')dy'}{\frac{1}{2}v^2(y)}$$

$$M_{eq}(y) = \frac{\frac{1}{2}\rho A(-\omega^2)\int_0^l u^2(y')dy'}{\frac{1}{2}(-\omega^2)u^2(y)} = \frac{\rho wt \int_0^l [X_{\mathrm{mod}e}(y')]^2 dy'}{[X_{\mathrm{mod}e}(y)]^2}$$

Xmode is the **”shape” of the fundamental mode**

= displacement as a function of y

$$X_{\text{mode}} = \text{shape of the fundmental mode}$$
$$= \text{displacement as a function of y}$$
$$X_{\text{mode}}(y) = \xi(\cos\beta y - \cosh\beta y) + (\sin\beta y - \sinh\beta y)$$
$$\beta = 4.730 / L_r, \text{"wave number"}$$
$$\xi = -1.01781$$

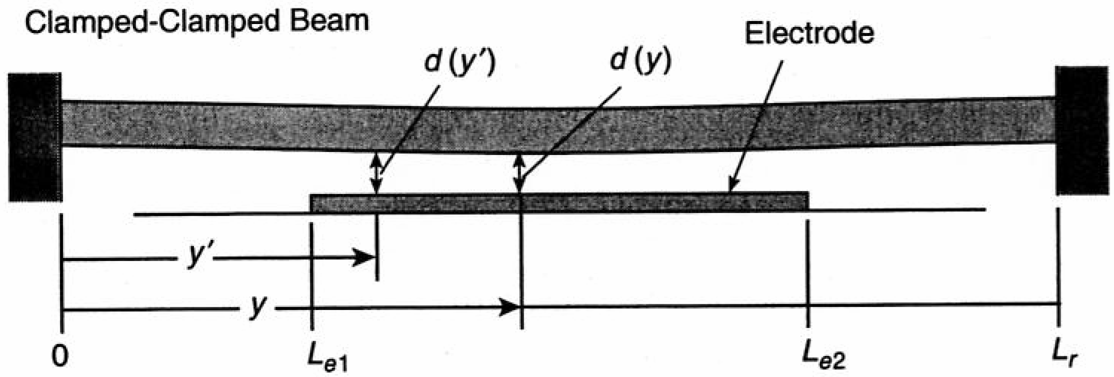


Figure 12.9. Resonator cross-sectional schematic for frequency-pulling and impedance analysis.

Procedure, contd.

- J) After calculation of the equivalent mass as function of (y), the equivalent spring stiffness k_r (y) and damping factor c_r (y) can be calculated
  - k_r = "equivalent", eg. influenced both by mechanical and electrical effects

Resonance frequency

$$f_0 = \frac{1}{2\pi}\sqrt{\frac{k_r}{m_r}}, \quad \omega_0^{\ 2} = \frac{k_r}{m_r}$$

J.

Equivalent spring stiffness

$k_r(y) = \omega_0^{\ 2} \cdot m_r(y)$, where $m_r(y)$ is the equivalent mass

The damping factor $c_r(y)$:

$$s^2 + \frac{b}{m}s + \frac{k}{m} = s^2 + \frac{\omega_0}{Q}s + \omega_0^{\ 2}$$

$$c = m\frac{\omega_0}{Q} = \frac{m\sqrt{k/m}}{Q} = \frac{\sqrt{km}}{Q}$$

By just looking at the **mechanical contribution**:
A certain frequency, ω_nom, and a corresponding Q-factor, Q_nom are obtained:

The mechanical spring constant : $k_m(y)$

gives the nominal values : $\omega_{nom},\ Q_{nom}$

The damping is only dependent on the mechanical factors :

$$c_r(y) = b = \frac{\sqrt{k_m(y)\cdot m_r(y)}}{Q_{nom}},\ \text{where}\ k_m(y) = \omega_{nom}{}^2 \cdot m_r(y)$$

K.

$$c_r(y) = \frac{\omega_{nom}\cdot m_r(y)}{Q_{nom}} = \frac{k_m(y)}{\omega_{nom} Q_{nom}}$$

Q_nom is the Q-factor of the resonator without the effect of the applied voltage
$k_m(y)$ is the mechanical stiffness without being influenced by the applied voltage and electrodes

Tunable electrical spring stiffness

- Spring stiffness can be tuned by Vp
  - The result depends on ratio between k_e and k_m

- L) Calculate how k_e depends on position y

The resonance frequency can be tuned by Vp

The electrically tunable spring constant, k_e, is subtracted from the mechanical one

The electrostatic beam - softening will change the spring stiffness

The resulting spring constant will be decreased :

$k_r = k_m - k_e$, mechanical minus electrical

The resonance frequency

$$f_0 = \frac{1}{2\pi}\sqrt{\frac{k_m - k_r}{m_r}} = \frac{1}{2\pi}\sqrt{\frac{k_m}{m_r}(1 - \frac{k_e / m_r}{k_m / m_r})} = \frac{1}{2\pi}\sqrt{\frac{k_m}{m_r}}(1 - \langle\frac{k_e}{k_m}\rangle)^{1/2}$$

$$f_0 = 1.03\chi\sqrt{\frac{E}{\rho}} \cdot \frac{h}{L^2}(1 - \langle\frac{k_e}{k_m}\rangle)^{1/2}$$

The relation is changed along the y-direction and has to be ”summed” in an integral

k_e is dependent on the capacitance $C(y')$ which is dependent on the gap $d(y')$ caused by V_P

By equating the potential energy to the work :

(integration of "the Hookes law-force" times distance for a parallel plate C)

$$U = \frac{1}{2}k_e \cdot d^2 = \frac{1}{2}CV_P^{\ 2} = \frac{1}{2}V_P^{\ 2}\frac{\varepsilon_0 A}{d}$$

$$k_e = V_P^{\ 2}\frac{C}{d^2} = V_P^{\ 2}\frac{\varepsilon_0 A}{d^3}$$

L. A contribution to the total spring stiffness from an element at the location y' and with a small electrode width dy'

$$dk_e(y') = V_P^{\ 2}\frac{\varepsilon_0 W_r dy'}{[d(y')]^3}$$

The local spring stiffness is dependent on the gap!

(d is the displacement from an equilibrium position)

The gap, d(y), has to be computed:

A force of F will give a displacement, d, from the equilibrium position where $V_P = 0$:

$$F = \frac{1}{2} V_P^{\ 2} \frac{\varepsilon_0 A}{d^2} = k \cdot \text{"displacement"}$$

(at each point, y)

$$d(y) = d_0 - \frac{1}{2} V_P^{\ 2} \varepsilon_0 W_r \int_{L_{e1}}^{L_{e2}} \frac{1}{k_m(y')[d(y')]^2} \cdot \frac{X_{sh}(y)}{X_{sh}(y')} dy'$$

The equation must be solved iteratively

Static bending shape due to the distributed DC force

When $d(y)$ has been found, then $dk_e(y')$ can be computed:

$$dk_e(y') = V_P^{\ 2} \frac{\varepsilon_0 W_r dy'}{[d(y')]^3}$$

Then

$$\langle \frac{k_e}{k_m} \rangle = g(d, V_P) = \int_{L_{e1}}^{L_{e2}} \frac{dk_e(y')}{k_m(y')} dy'$$

Simplification **(De Los Santos):**

Assume that the beam is flat over the electrode

Potential energy

$$U_1 = \frac{1}{2} C V_P^2$$

Work being done to move the beam a distance g AGAINST the force due to the electrical beam stiffness k_e
(The spring stiffness is now considered to be CONSTANT in each pont y´)

$$U_2 = \int_0^g k_e \cdot x \cdot dx = \frac{1}{2} k_e \cdot g^2$$

The energies can be set equal

$$\frac{1}{2} k_e \cdot g^2 = \frac{1}{2} C \cdot V_P^2$$

Simplified expression for the electrical beam stiffness

$$k_e = \frac{C \cdot V_P^2}{g^2}$$

Simplified expression for frequency

$$f=\frac{1}{2\pi}\sqrt{\frac{k_m-k_e}{m_r}}=\frac{1}{2\pi}\sqrt{\frac{k_m}{m_r}\left(1-\frac{k_e}{k_m}\right)}$$

$$=\frac{1}{2\pi}\sqrt{\frac{k_m}{m_r}}\cdot\sqrt{1-\frac{k_e}{k_m}}=f_{nom}\cdot\sqrt{1-\frac{C\cdot V_P{}^2}{k_m\cdot g^2}}$$

Substitute for C: $C=\varepsilon_0\cdot\frac{A}{g}$

$$f=f_{nom}\cdot\sqrt{1-\frac{\varepsilon_0\cdot A\cdot V_P{}^2}{k_m\cdot g^3}}$$

This is equivalent to the previous calculations

$$k_e = \varepsilon_0 \cdot \frac{A \cdot V_P{}^2}{g^3}$$

$$dk_e(y') = V_P{}^2 \cdot \frac{\varepsilon_0 \cdot W_r \cdot dy'}{[d(y')]^3}$$

Differential electrical spring stiffness in location y´ and with an electrode width dy´

Beam-softening

- Resonance frequency decreases by

$$\sqrt{1 - C_0 \cdot V_P{}^2 / (k_m \cdot g^2)}$$

  - → **resonance frequency may be tuned electrically!**

$$f_o = \frac{1}{2\pi}\sqrt{\frac{k_r}{m_r}} = 1.03\kappa\sqrt{\frac{E}{\rho}\frac{h}{L_r^2}}\,[1 - g(V_P)]^{1/2}, \qquad (12.2)$$

Small signal equivalent

- An electrical equivalent circuit is needed to model and simulate the impedances of this micro-mechanical resonator in a **common** electromechanical circuit

$$L_x = \frac{m_{re}}{\eta_e^2}, \qquad C_x = \frac{\eta_e^2}{k_{re}}, \qquad R_x = \frac{\sqrt{k_{re} m_{re}}}{Q\eta_e^2} = \frac{C_{re}}{\eta_e^2}, \qquad (12.17)$$

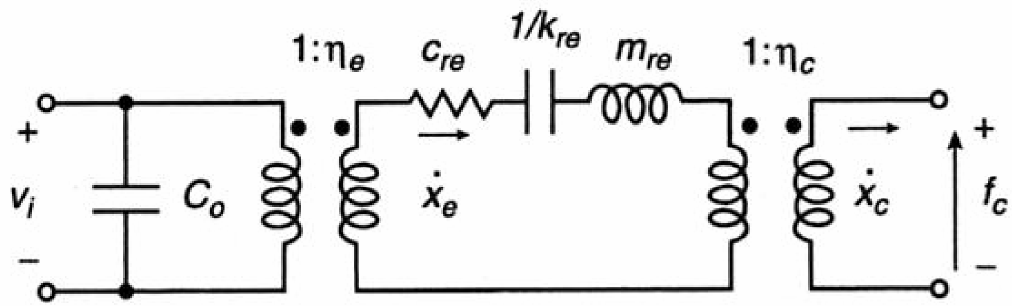


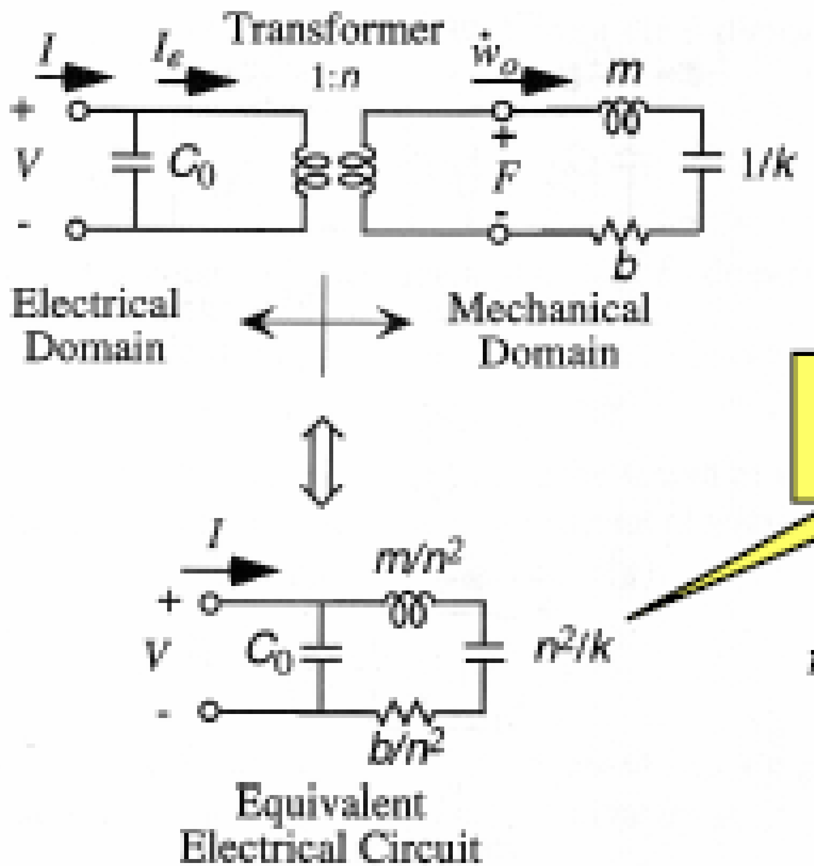
Figure 12.10. Equivalent circuit for a μmechanical resonator with both electrical (voltage v_i) and mechanical (force f_c) inputs and outputs.

Coupling coefficient

- Look into the circuit from the left side
- Observe a transformed LCR-circuit with new element values given by (12.17)
  - Electromechanical coupling coefficient = ”transformer turns ratio”

- Coupling coefficient is calculated in notes from UCLA
  - Discussed in relation to 2-port lateral comb-drive actuator (L10)

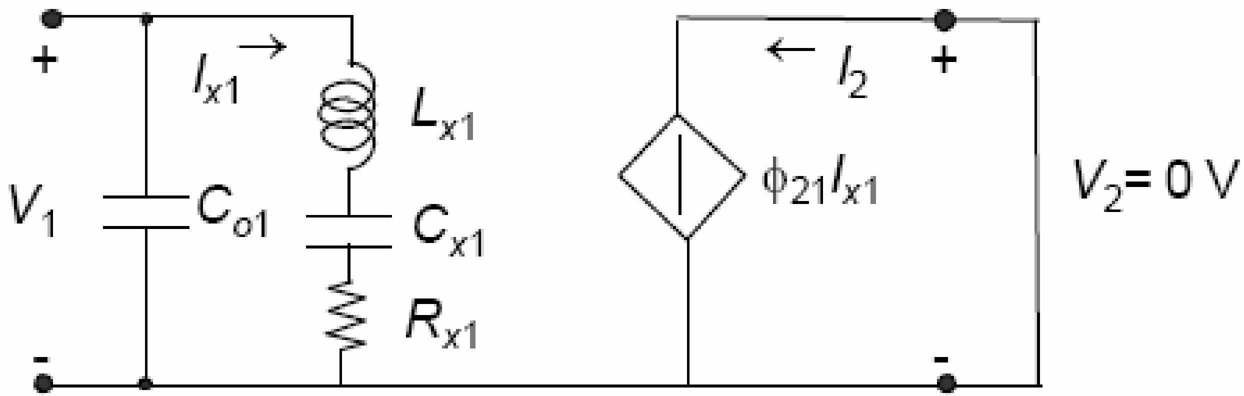
Small Signal Equivalent Circuit of Microresonators



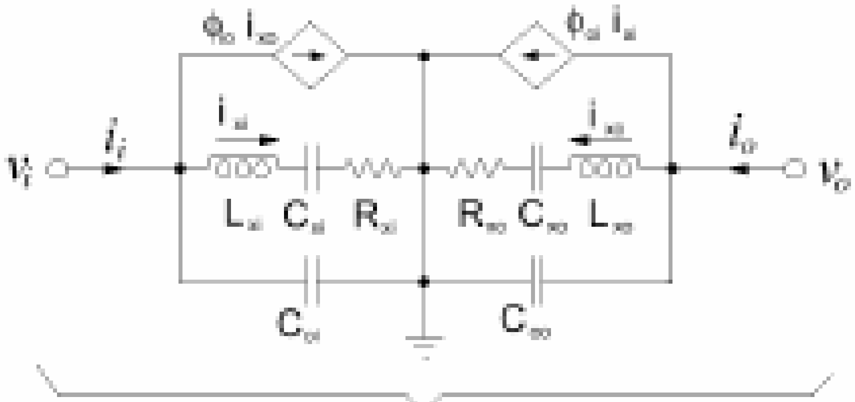
Unit of n^2/k is Farad

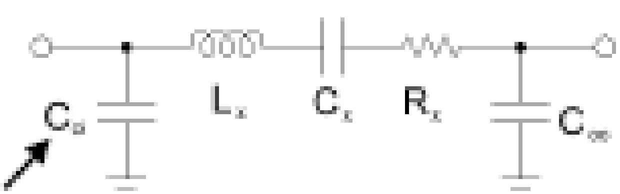
$$n = V_{dc} \frac{\partial C}{\partial x}$$

Two-Port Equivalent Circuit ($v_2 = 0$)



Equivalent Circuit of 2-Port Resonator
(in Electrical Domain)





$C_x = 0.5$ fF

$L_x = 200$ kH

$R_x = 500$ kΩ

$C_{oi}, C_{oo} = 15$ fF

Fixed electrical Capacitance Between fixed comb And ground plane

$$C_{xn} = \frac{\eta_n^2}{k} \qquad R_{xn} = \frac{\sqrt{km}}{Q\eta_n^2} \qquad \eta_n = V_{Pn}\frac{\partial C_n}{\partial x}$$

$$L_{xn} = \frac{m}{\eta_n^2} \qquad \phi_{mn} = \frac{\eta_m}{\eta_n}$$

C. T.-C. Nguyen, "Micromechanical resonators for oscillators and filters," *Proceedings* IEEE International Ultrasonics Symposium, Seattle, WA, pp. 489-498, Nov. 7-10, 1995

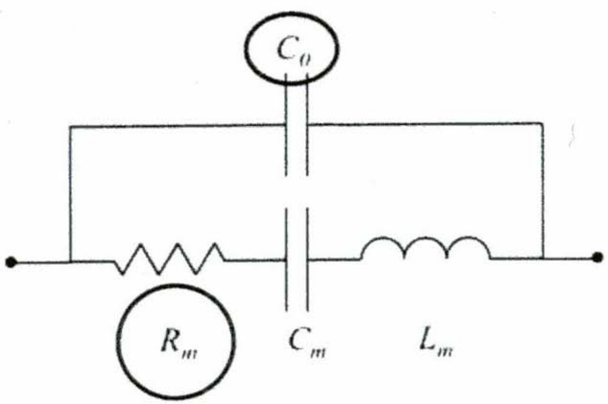
Discussion:

FSRM

Resonator equivalent circuit

Two types of currents possible:

- **from resonator motion** (should dominate!)
- from electrodes and resonator acting as pure electrical structure (from feedthrough capacitance)



Admittance at resonance is

$$Y_{in} = \frac{1}{R_m} + j\omega_o C_o$$

where we want to minimize the motional resistance, R_m :

$$R_m = \frac{\sqrt{k * m}}{Q\eta^2} \qquad \eta = V_{DC}\frac{dC}{dg}$$

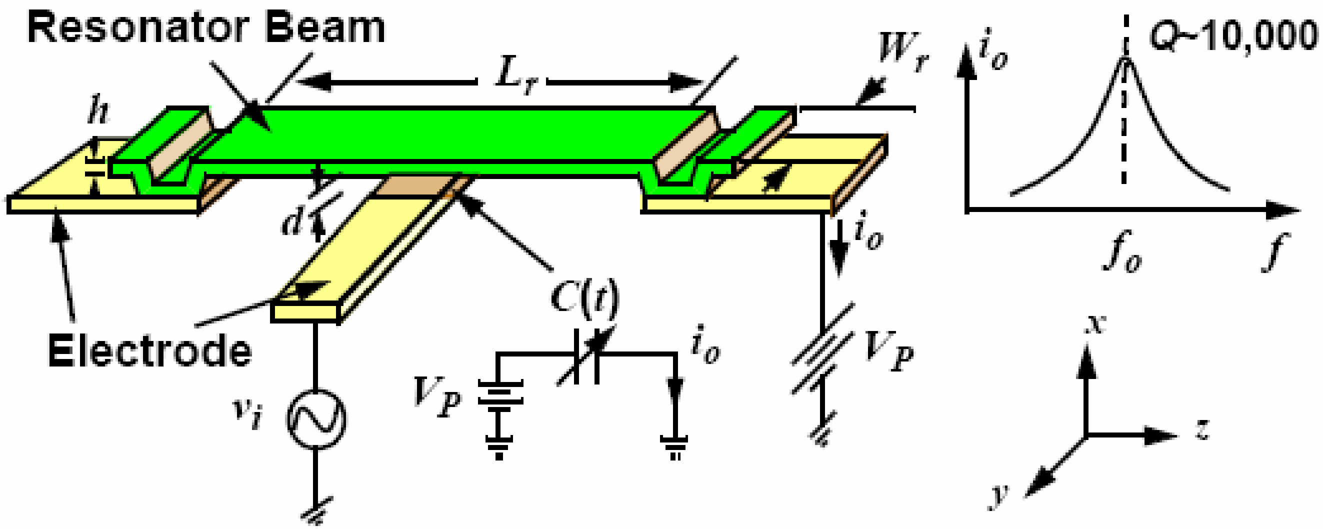
- Need:
  - ➢High Q
  - ➢High coupling (high voltage or small gap)
  - ➢Low mass
  - ➢Low stiffness (!)

95

Ionescu, EPFL

Vertically-Driven Micromechanical Resonator

- To date, most used design to achieve VHF frequencies



$$f_o = \frac{1}{2\pi}\sqrt{\frac{k_r}{m_r}} = 1.03\sqrt{\frac{E}{\rho}}\frac{h}{L_r^2}$$

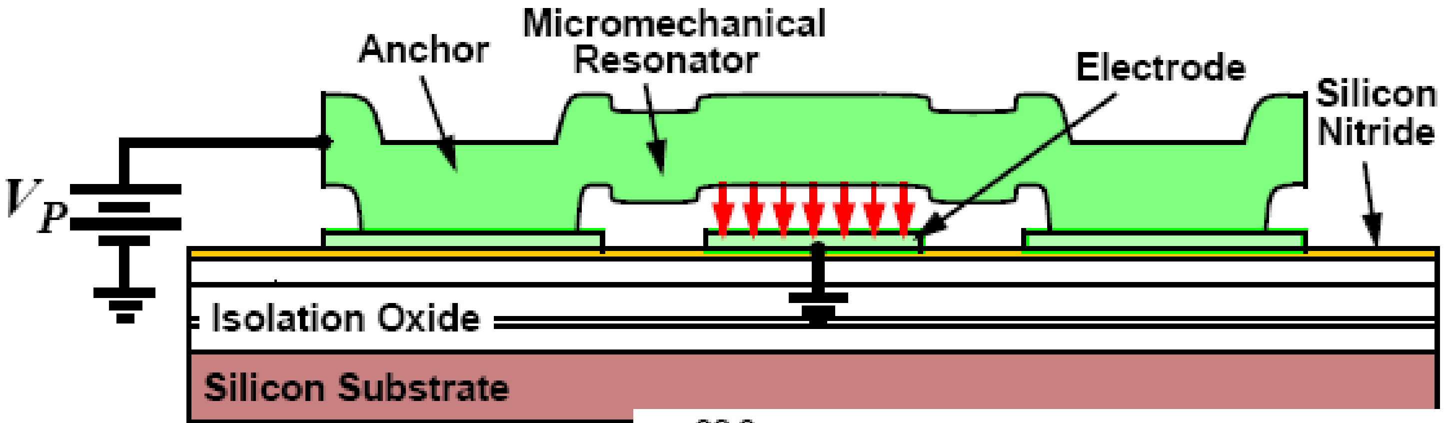
(e.g. m_r=10^{-13}kg)

E = Youngs Modulus
ρ=density

- Smaller mass ➠ higher frequency range and lower series R_x

C. T.-C. Nguyen — Univ. of Michigan

Voltage-Controllable Center Frequency



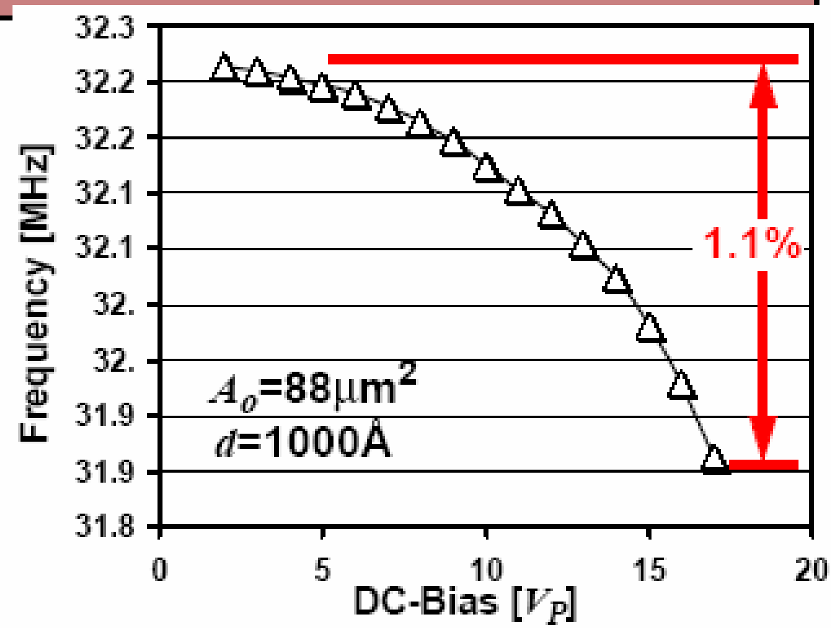
- Quadrature force ➠ voltage-controllable electrical stiffness:

$$k_e = \frac{\varepsilon_o A_o}{d^3} V_P^2$$

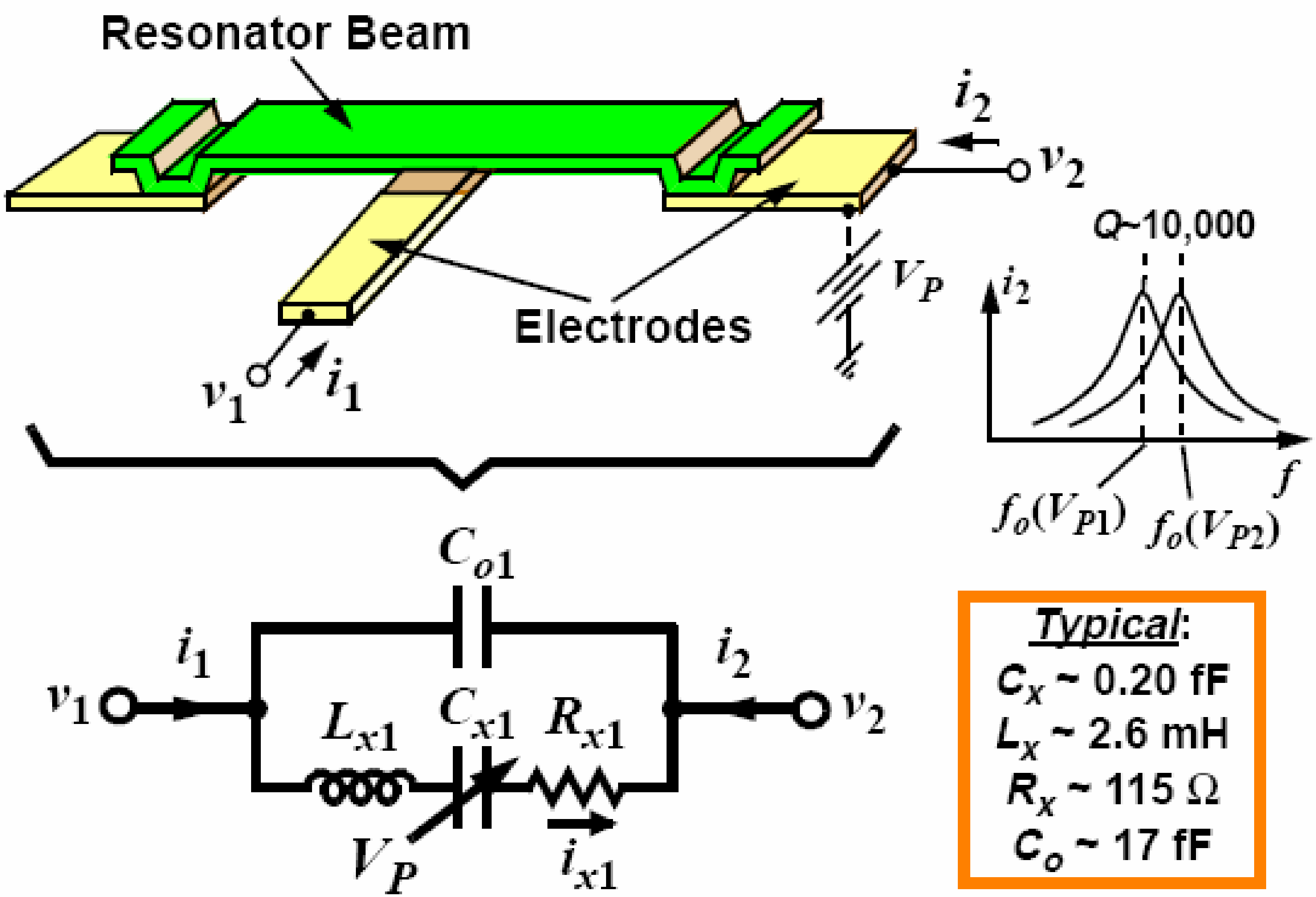
Electrode Overlap Area (A_o)

Finger Gap (d)

$$f_o = \frac{1}{2\pi}\sqrt{\frac{k_m - k_e}{m_r}}$$



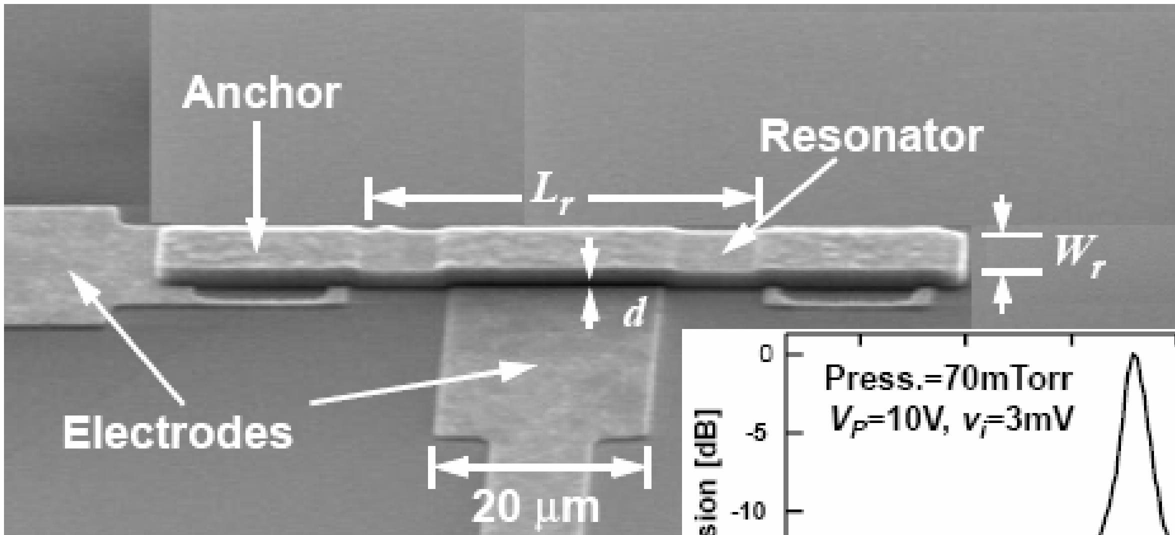
Micromechanical Resonator Equivalent Circuit

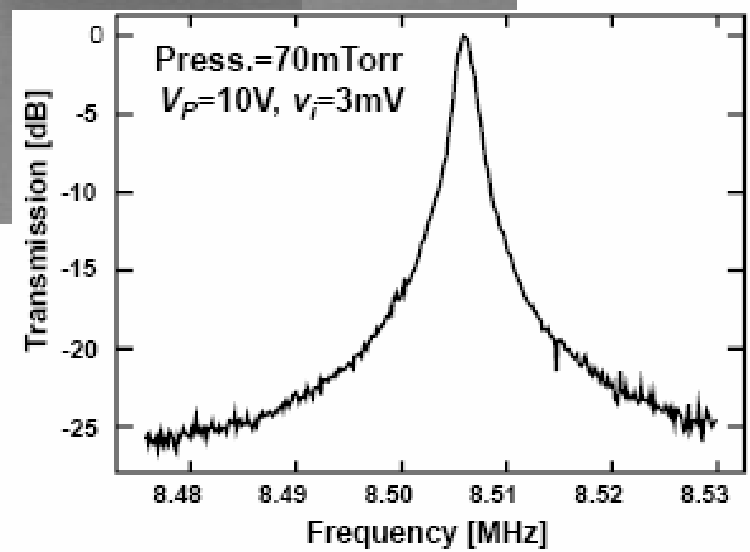


Typical:
C_x ~ 0.20 fF
L_x ~ 2.6 mH
R_x ~ 115 Ω
C_o ~ 17 fF

Fabricated HF μMechanical Resonator

- Surface-micromachined, $POCl_3$-doped polycrystalline silicon





L_r=40.8 μm, W_r=8 μm, h=2 μm, d=0.1μm

- Extracted Q = 8,000 (vacuum)
- Freq. influenced by dc-bias and anchor effects

Loss, c-c-beam

- Resonance frequency increases when the stiffness of a beam increases
  - Also: More energy pr. cycle enters the substrate via the anchors

- c-c-beam has loss through anchors
  - → Q-factor decreases when frequency increases
  - c-c-beam is not the best structure for high frequency!
  - Ex. Q = 8,000 at 10 MHz, Q = 300 at 70 MHz
- c-c beam may be used as a reference oscillator or HF/VHF filter/mixer

- **Use of ”free-free beam” can reduce the energy loss via anchors to the substrate!**

free-free-beam

- Beneficial for reducing loss to substrate via anchors
- f-f-beam is suspended using 4 support-beams in width-direction
  - Torsion-support
  - Anchoring at nodes for ”flexural mode”
- Support dimension is a quarter-wavelength of f-f-beam resonance frequency
    - The electrical impedance at the flexural nodes is then infinite
    - Beam vibrates without energy loss as if there is no support
- Higher Q is achieved
  - Ex. Q= 20,000 at 10 – 200 MHz
  - Applied in reference-oscillators, HF/VHF-filter/mixer

free-free beam

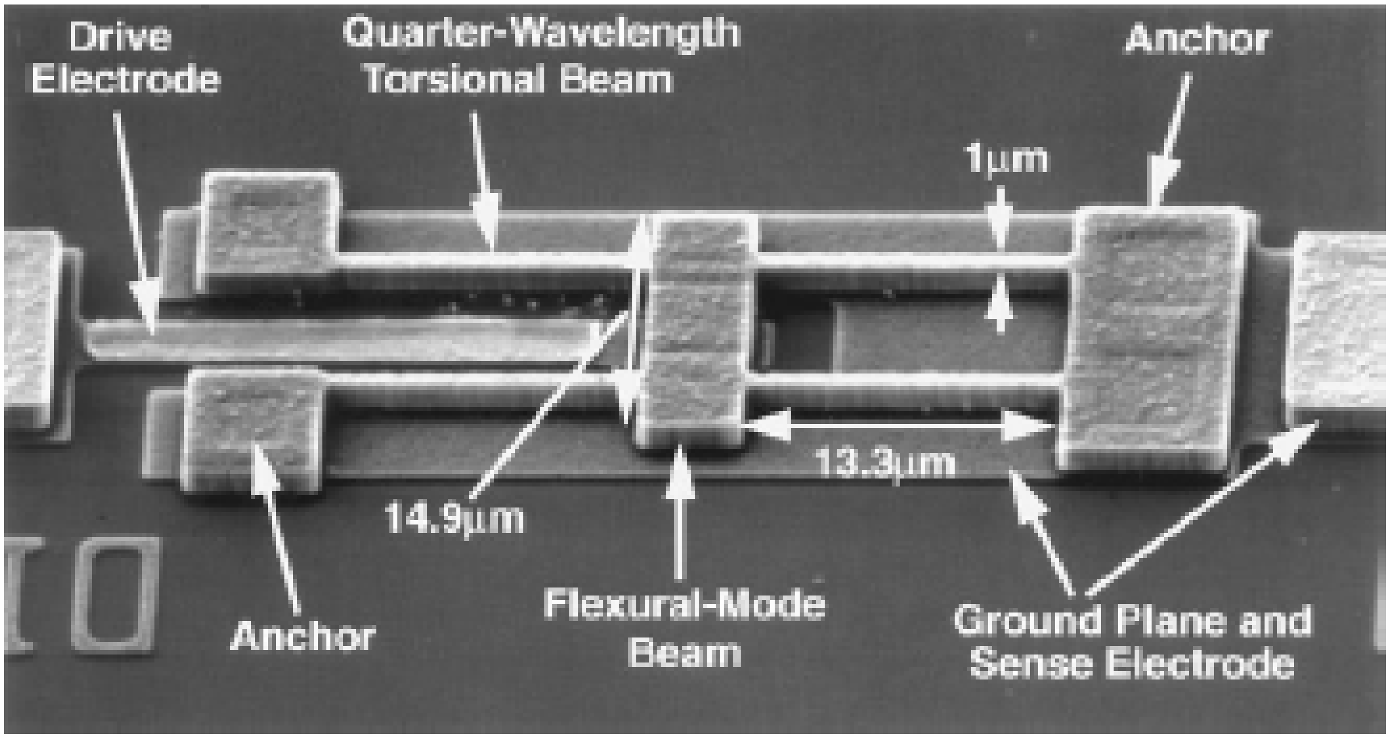
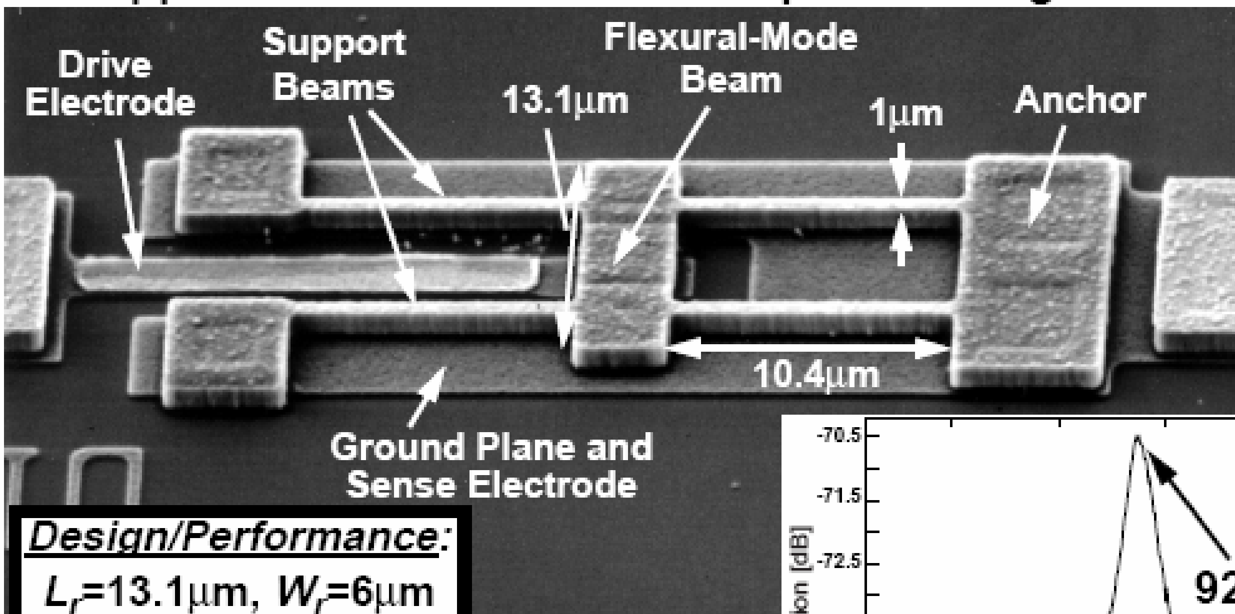


Fig. 29. SEM of free–free beam virtually levitated micromechanical resonator with relevant dimensions for $f_o = 71$ MHz.

Nguyen, 1999

92 MHz Free-Free Beam μResonator

- Free-free beam μmechanical resonator with non-intrusive supports ➠ reduce anchor dissipation ➠ higher Q



Design/Performance:

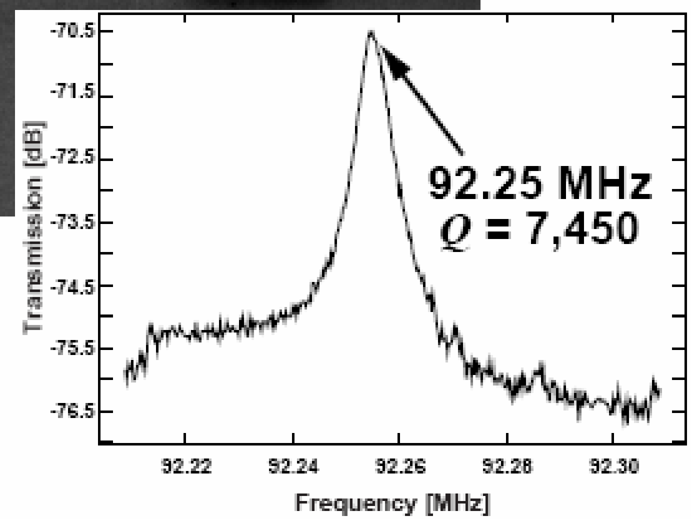
L_r=13.1μm, W_r=6μm

h=2μm, d=1000Å

V_P=28V, W_e=2.8μm

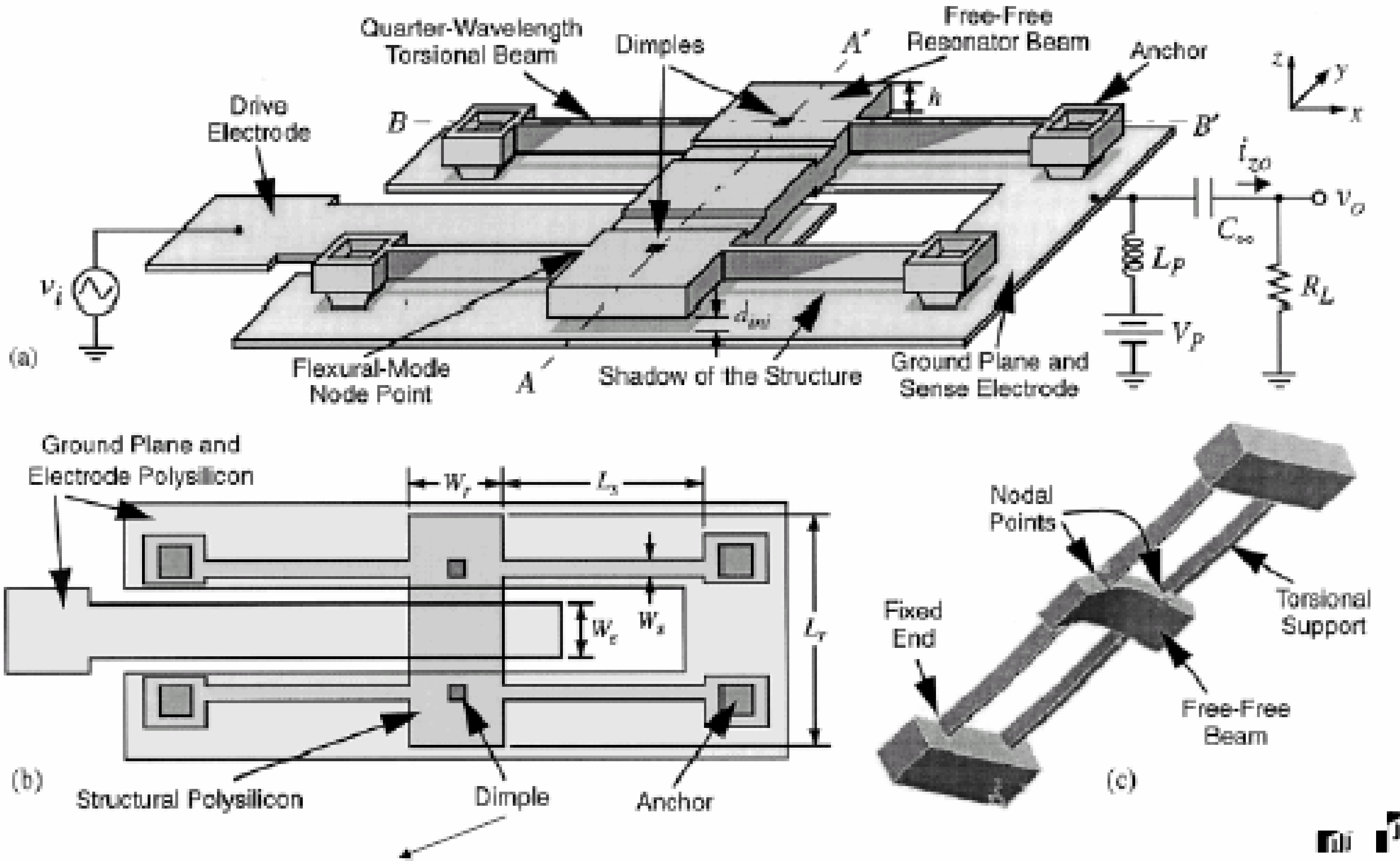
f_o~92.25MHz

Q~7,450 @ 10mTorr



[Wang, Yu, Nguyen 1998]

VHF Free-Free Beam High-Q Micromechanical Resonator

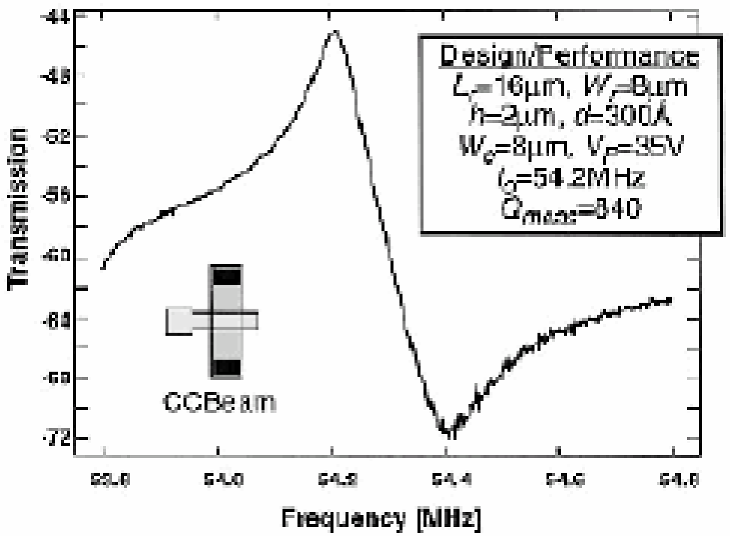


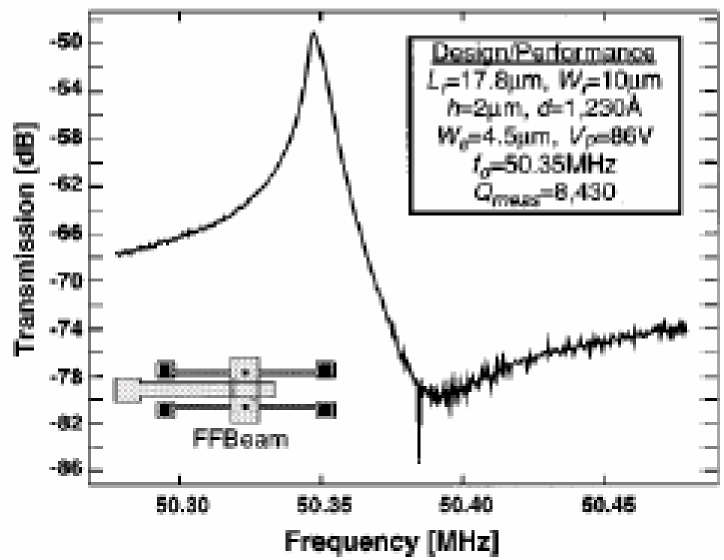
(determined the gap)

M. C. Wu

J. MEMS, Vol. 9, No. 3, 2000, C. T. –C. Nguyen, et al.

Comparison of Frequency Characteristics





Clamped-clamped beam

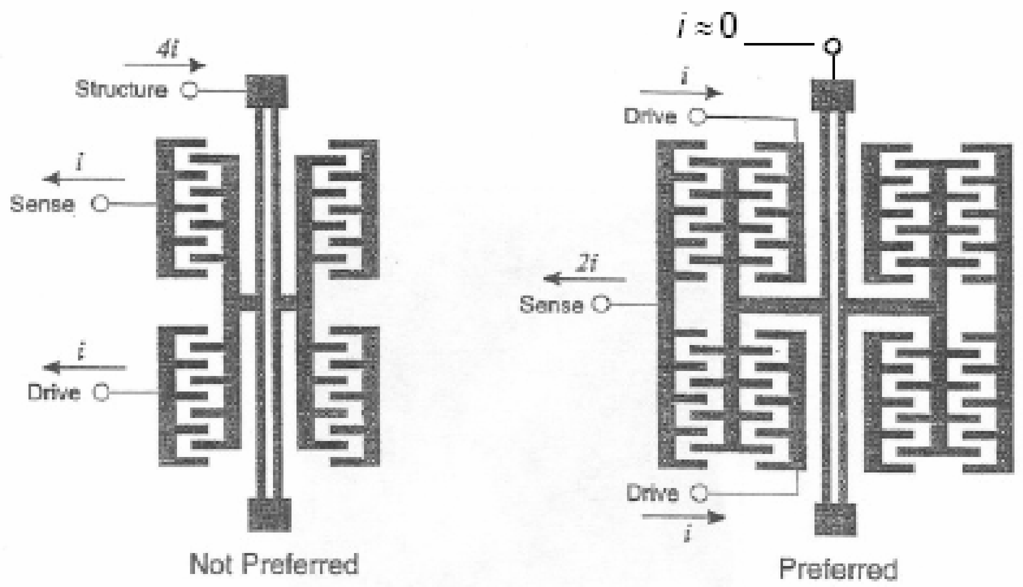
- L_r=16 μm, d=0.03 μm
- V_p=35 V, f_0=54.2 MHz
- Q=840

Free-free beam

- L_r=17.8 μm, d=0.12 μm
- V_p=86 V, f_0=50.35 MHz
- Q=8,430

Other resonator types

Double-Ended Tuning Fork Resonators



Current through structure → more resistance (decreases Q)
more feedthrough to substrate

EE C245 – ME C218 Fall 2003 Lecture 26

T. Roessig, Ph.D.,ME ,
UC Berkeley, 1997

18

"Tuning fork" → balanced!

Scaling of Lateral Micromechanical Resonators

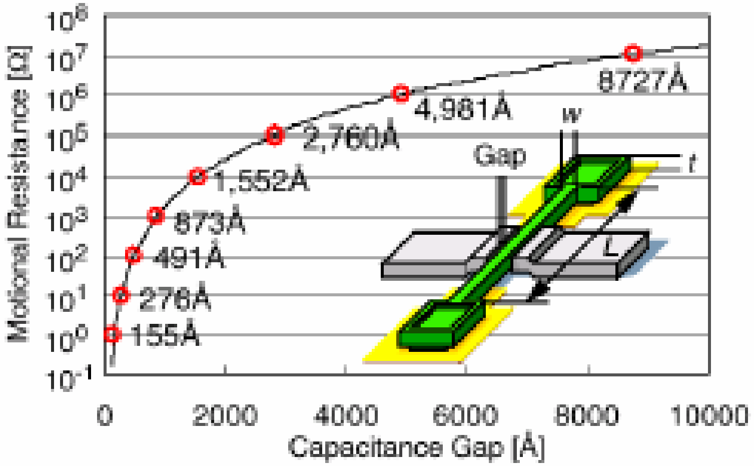


Fig. 1: Simulated plot of motional resistance versus electrode-to-resonator gap for a 40μm-long, 2μm-wide, 3μm-thick, lateral clamped-clamped beam μmechanical resonator.

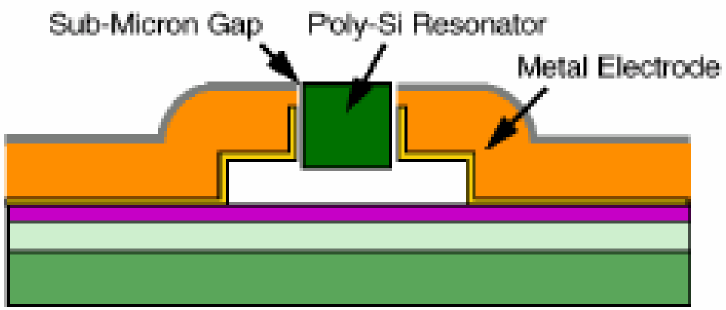
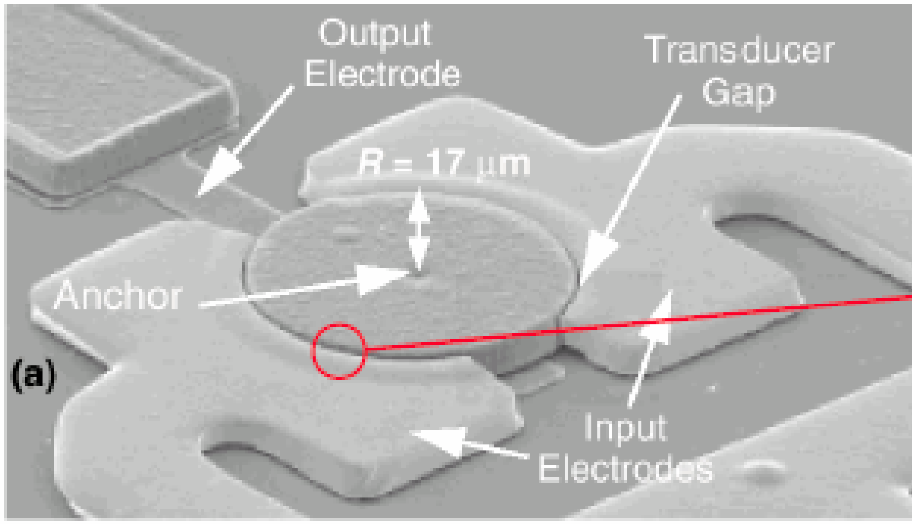


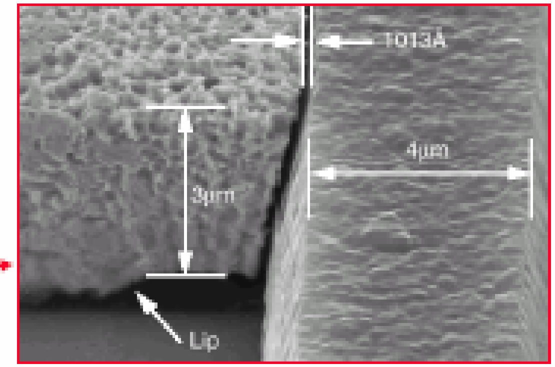
Fig. 2: Cross-section of the described sub-μm electrode-to-resonator gap process for lateral μstructures with metal electrodes.

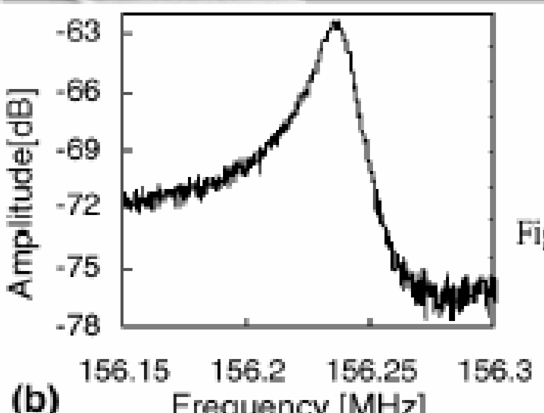
- Advantages of lateral resonator
  - Wider variety of resonant modes
  - Balanced resonators (push-pull)
  - More design flexibility
- As frequency scales up
  - Resonator size shrinks
  - Capacitive transducer gaps must also shrink (to sub-100 nm for VHF)
  - High aspect ratio structures
- Combine Poly-Si (high-Q structural materials) with metal electrode (high conductivity)
  - Self-aligned process

Hsu, Clark, Nguyen, "A sub-micron capacitive gap process for multiple-metal-electrode lateral micromechanical resonators," MEMS 2001, p. 349

Radial Contour-Mode Disk μ-mechanical Resonator







Data:
R=17μm, h=2μm
d=1,000Å, V_P=35V
f_o=156.23MHz, Q=9,400

Fig. 5: SEM and measured frequency characteristic for a 156.23 MHz contour-mode disk μmechanical resonator fabricated via the process of Fig. 3.

- Radial contour mode allows high resonant frequency without requiring sub-micron structures
- Place anchor at disk center – nodal point of contour mode
  ➔ Reduce mechanic loss and increase Q

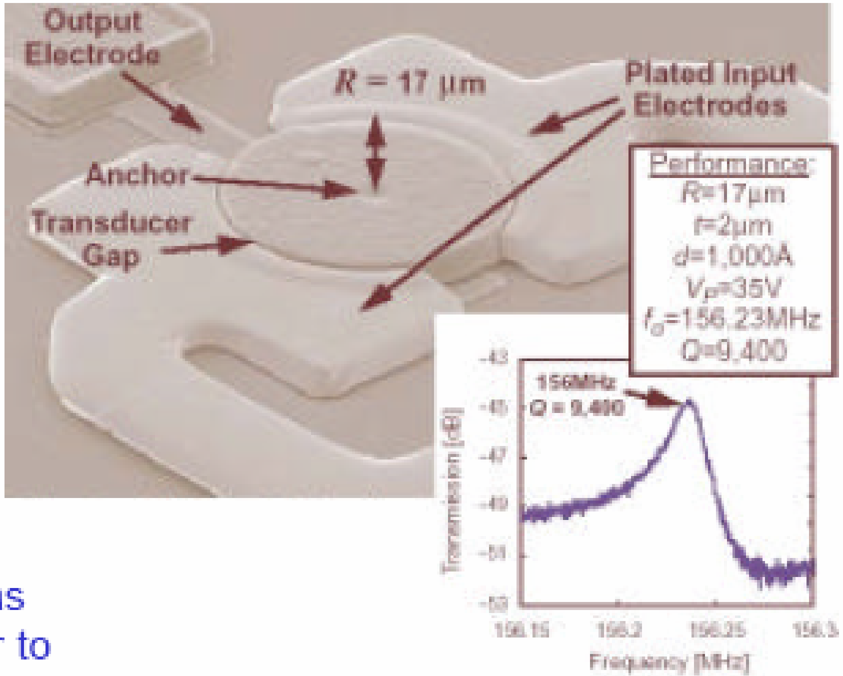
Hsu, Clark, Nguyen, "A sub-micron capacitive gap process for multiple-metal-electrode lateral micromechanical resonators," MEMS 2001, p. 349

Disk resonators

- Advantages of using disks compared to beams
  - Reduced air damping
    - Vacuum not needed to measure Q-factor
  - Higher stiffness
    - Higher frequency for given dimensions
  - Larger volume
    - Higher Q because more energy is stored
    - Less problems with thermal noise
- Periphery of the disk may have different motional patterns
  - Radial, wine-glass

Increasing the Resonant Frequency

option 2. spring rate → ∞

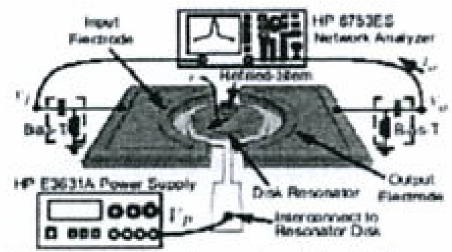


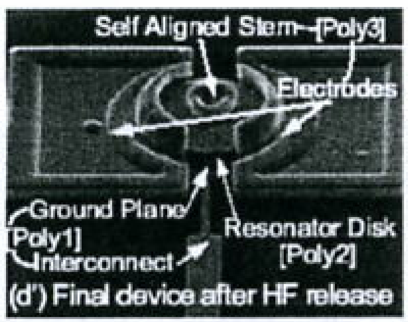
Clark Nguyen, Michigan

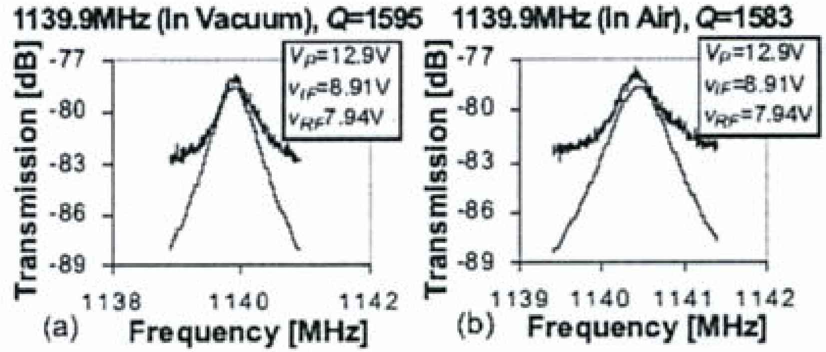
Motivation: keep mass as large as possible in order to improve precision of fab, power handling

IEEE IEDM 2000.

Bulk contour-mode resonators







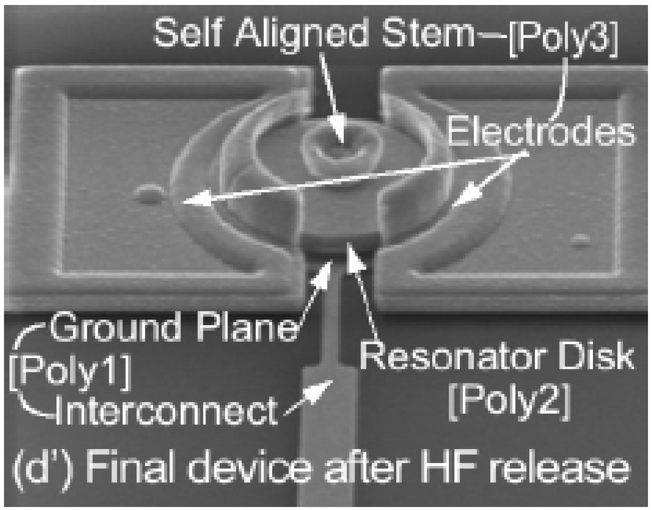
- > **1GHz** resonance frequency demonstrated
- **Q > 1'500** in both vaccum and air
- **Tcoeff ~-15ppm/°C**

J. Wang et al, Transducers 2003.

- **Bulk acoustic mode resonators / contour-mode disk resonators**
- **Frequency range: tens of kHz to GHz**
- **Quality factors > 10'000 for single crystal silicon demonstrated**
- **Further developments: process with nano-gaps → GHz frequency**

100

1.14 GHz Poly-Si Disk Resonator



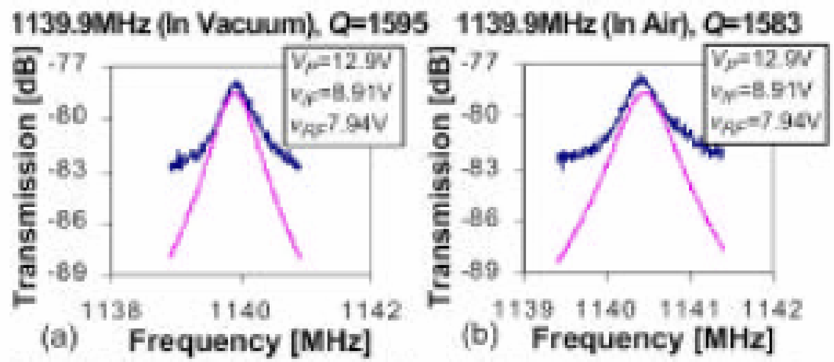


Fig. 7: Measured (dark) and predicted (light) frequency characteristics for a 1.14-GHz, 3rd mode, 20μm-diameter disk resonator measured in (a) vacuum and (b) in air, using a mixing measurement setup.

* Note *Q* in vacuum and in air is the same: little energy loss to ambient; however, energy loss through anchor ("stem") is significant
* EAM-like technique is used to extract the motional current.

Transducers '03, Boston

EAM = Electromechanical Amplitude Modulation (sinus also on "shuttle")

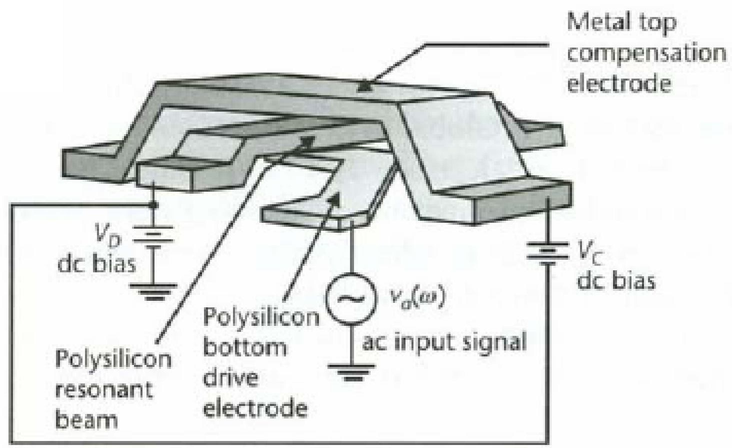
Limitations of micromechanical resonators

- Frequency-limitations
  - By reducing m to obtain higher frequency:
  - This will give fluctuations in frequency
    - ”mass loading”: interchange of molecules with environment
    - Air gas molecules have Brownian motion
- Energy limitations
  - Q depends on energy loss caused by damping
    - Viscous damping
    - Vertical motion: squeezed-film damping
    - Horizontal motion: slide film damping, Stokes- or Couette-type damping

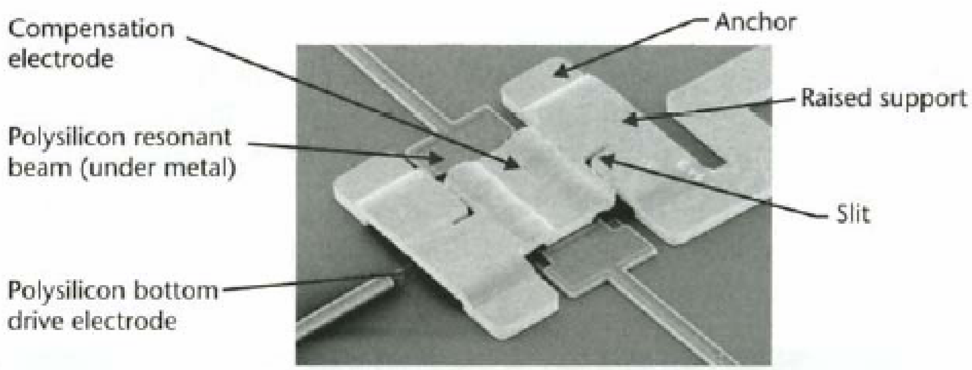
Limitations, contd.

- Temperature dependence
  - Resonance frequency changes due to temperature and aging
  - Increased temperature gives frequency decrease
    - Analog or digital compensation (feedback)
    - Mechanical compensation
      - Exploit structures with both compressive and tensile stress: opposing effects →

Temperature compensation



(a)



(b)

Figure 7.11 Illustration of the compensation scheme to reduce sensitivity in a resonant structure to temperature. A voltage applied to a top metal electrode modifies through electrostatic attraction the effective spring constant of the resonant beam. Temperature changes cause the metal electrode to move relative to the polysilicon resonant beam, thus changing the gap between the two layers. This reduces the electrically induced spring constant opposing the mechanical spring while the mechanical spring constant itself is falling, resulting in their combination varying much less with temperature. (a) Perspective view of the structure [23], and (b) scanning electron micrograph of the device. (*Courtesy of:* Discera, Inc., of Ann Arbor, Michigan.)

Temperature compensation, contd.

- Top-electrode **reduces** effective spring constant because Vc causes an electrostatic attraction
- Top-electrode will be elevated (gap increases) when the temperature increases → reduction of spring constant
- Generally the mechanical spring constant decreases by increased temperature. But the reduction will be less due to the effect of the top electrode (e.g. the ”beam-softening”-effect decreases)!